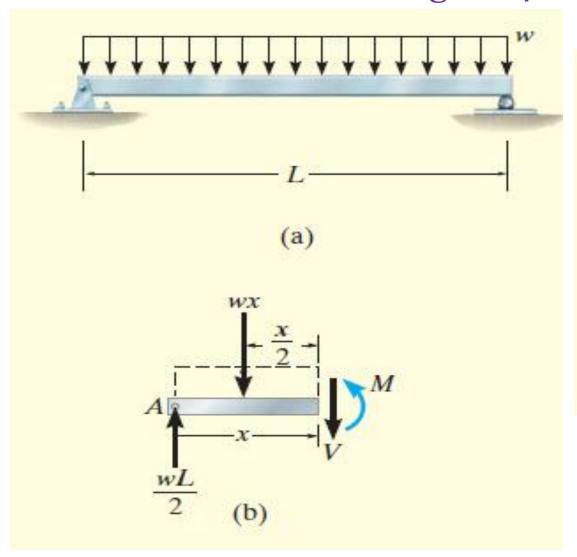
Example 1 Draw the shear and moment diagrams for the beam shown in Fig. 6–4a.



$$+ \uparrow \Sigma F_y = 0; \qquad \frac{wL}{2} - wx - V = 0$$

$$V = w \left(\frac{L}{2} - x\right) \tag{1}$$

$$M = \frac{w}{2}(Lx - x^2) \tag{2}$$

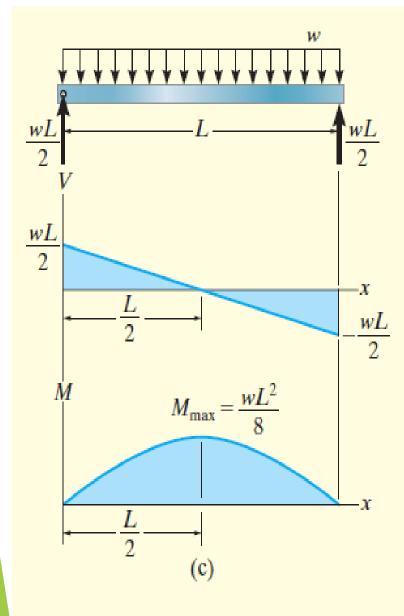


Fig. 6-4

Shear and Moment Diagrams. The shear and moment diagrams shown in Fig. 6–4c are obtained by plotting Eqs. 1 and 2. The point of zero shear can be found from Eq. 1:

$$V = w \bigg(\frac{L}{2} - x \bigg) = 0$$

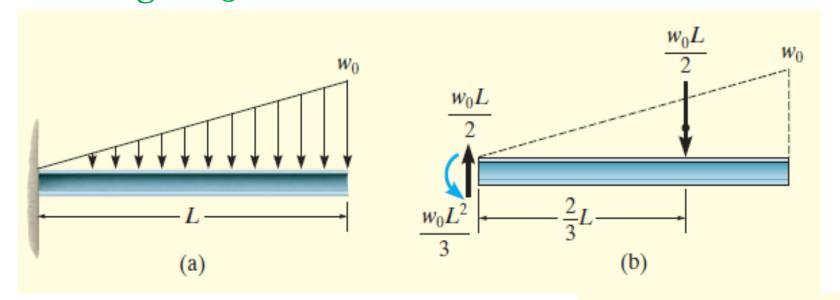
$$x = \frac{L}{2}$$

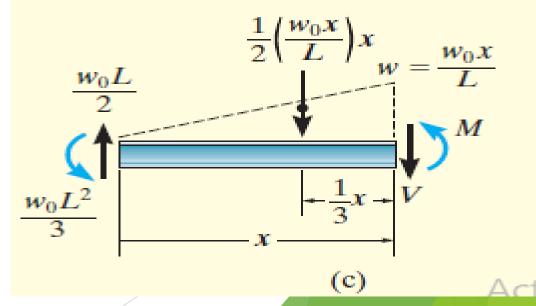
NOTE: From the moment diagram, this value of x represents the point on the beam where the *maximum moment* occurs, since by Eq. 6–2 (see Sec. 6.2) the *slope* V = dM/dx = 0. From Eq. 2, we have

$$M_{\text{max}} = \frac{w}{2} \left[L \left(\frac{L}{2} \right) - \left(\frac{L}{2} \right)^2 \right]$$

$$=\frac{wL^2}{8}$$

Example 2: Draw the shear and moment diagrams for the beam shown in Fig. 6–5a.





$$+\uparrow \Sigma F_y = 0;$$
 $\frac{w_0 L}{2} - \frac{1}{2} \left(\frac{w_0 x}{L}\right) x - V = 0$

$$V = \frac{w_0}{2L}(L^2 - x^2) \tag{1}$$

$$\zeta + \sum M = 0;$$

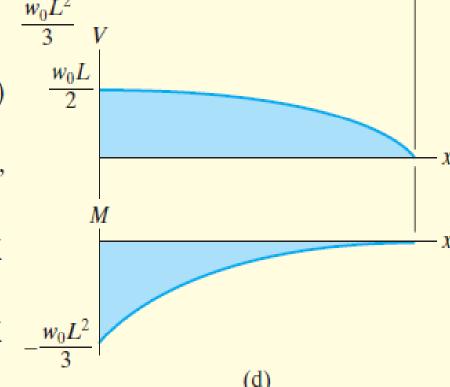
$$\frac{w_0 L^2}{3} - \frac{w_0 L}{2}(x) + \frac{1}{2} \left(\frac{w_0 x}{L}\right) x \left(\frac{1}{3}x\right) + M = 0 \quad \frac{w_0 L^2}{2}$$

$$M = \frac{w_0}{6L}(-2L^3 + 3L^2x - x^3) \tag{2}$$

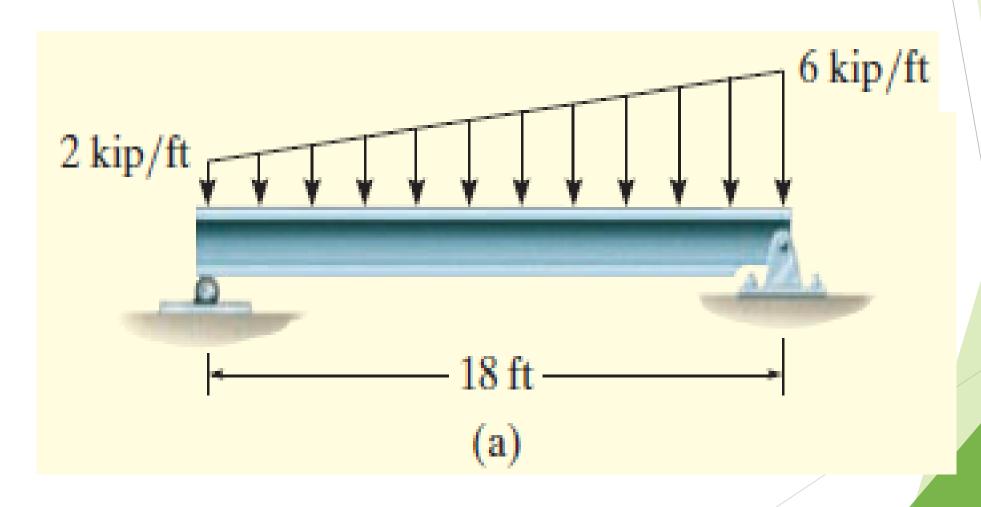
These results can be checked by applying Eqs. 6–1 and 6–2 of Sec. 6.2, that is,

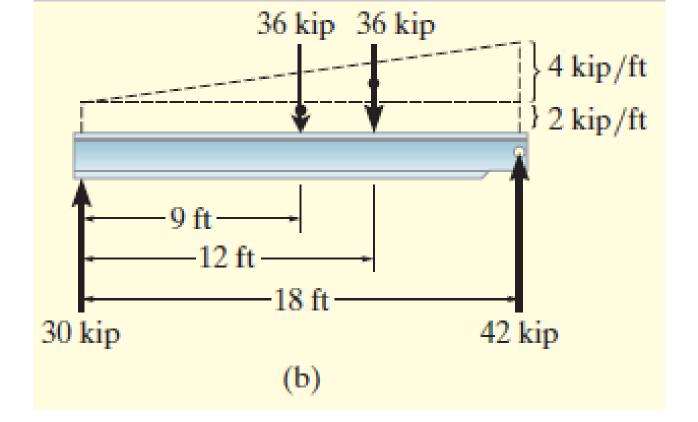
$$w = \frac{dV}{dx} = \frac{w_0}{2L}(0 - 2x) = -\frac{w_0x}{L}$$
 OK

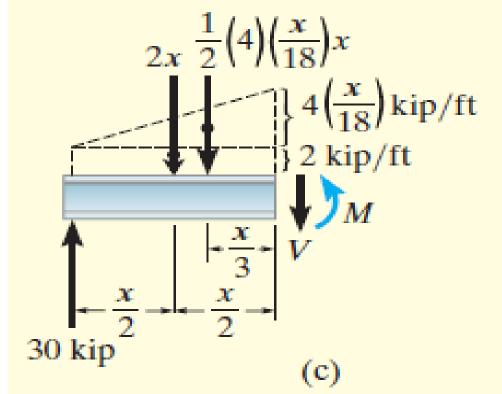
$$V = \frac{dM}{dx} = \frac{w_0}{6L}(0 + 3L^2 - 3x^2) = \frac{w_0}{2L}(L^2 - x^2)$$
 OK $-\frac{w_0L^2}{3}$



EXAMPLE 3
Draw the shear and moment diagrams for the beam shown in Figure.







$$+ \uparrow \Sigma F_y = 0; \ 30 \text{ kip} - (2 \text{ kip/ft})x - \frac{1}{2} (4 \text{ kip/ft}) \left(\frac{x}{18 \text{ ft}}\right)x - V = 0$$

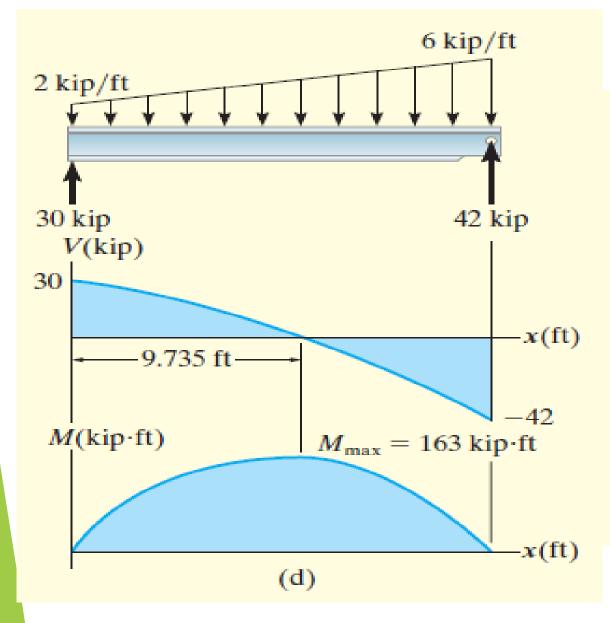
$$V = \left(30 - 2x - \frac{x^2}{9}\right) \text{kip}$$
 (1)

 $(+\Sigma M=0;$

$$-30 \, \text{kip}(x) + (2 \, \text{kip/ft}) x \left(\frac{x}{2}\right) + \frac{1}{2} (4 \, \text{kip/ft}) \left(\frac{x}{18 \, \text{ft}}\right) x \left(\frac{x}{3}\right) + M = 0$$

$$M = \left(30x - x^2 - \frac{x^3}{27}\right) \text{kip} \cdot \text{ft} \tag{2}$$

Equation 2 may be checked by noting that dM/dx = V, that is, Eq. 1. Also, $w = dV/dx = -2 - \frac{2}{9}x$. This equation checks, since when x = 0, w = -2 kip/ft, and when x = 18 ft, w = -6 kip/ft, Fig. 6–6a.



Shear and Moment Diagrams. Equations 1 and 2 are plotted in Fig. 6–6d. Since the point of maximum moment occurs when dM/dx = V = 0 (Eq. 6–2), then, from Eq. 1,

$$V = 0 = 30 - 2x - \frac{x^2}{9}$$

Choosing the positive root,

$$x = 9.735 \, \text{ft}$$

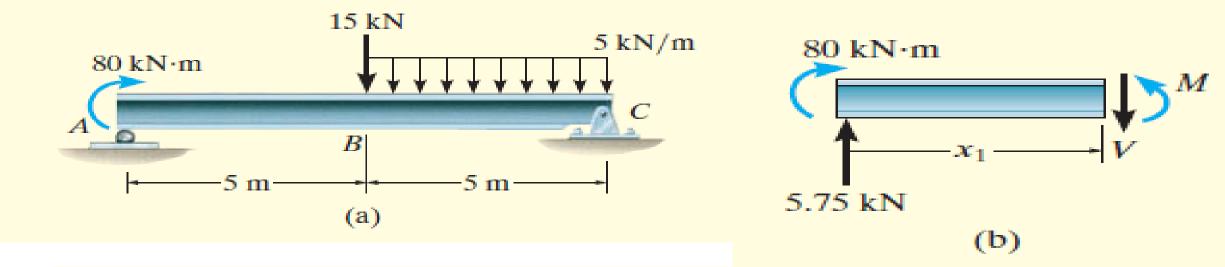
Thus, from Eq. 2,

$$M_{\text{max}} = 30(9.735) - (9.735)^2 - \frac{(9.735)^3}{27}$$

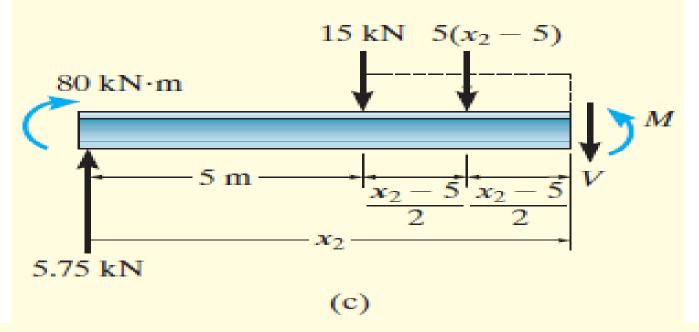
= 163 kip · ft

EXAMPLE 4

Draw the shear and moment diagrams for the beam shown in Figure.



$$0 \le x_1 < 5 \text{ m}$$
, Fig. 6–7b:
 $+ \uparrow \Sigma F_y = 0$; $5.75 \text{ kN} - V = 0$
 $V = 5.75 \text{ kN}$
 $(+ \Sigma M = 0)$; $-80 \text{ kN} \cdot \text{m} - 5.75 \text{ kN} x_1 + M = 0$
 $M = (5.75x_1 + 80) \text{ kN} \cdot \text{m}$



$$5 \text{ m} < x_2 \le 10 \text{ m}$$
, Fig. 6–7c:

$$+\uparrow \Sigma F_y = 0;$$
 5.75 kN $-$ 15 kN $-$ 5 kN/m($x_2 - 5$ m) $V = 0$
 $V = (15.75 - 5x_2)$ kN (3)

$$(+\Sigma M = 0; -80 \text{ kN} \cdot \text{m} - 5.75 \text{ kN} x_2 + 15 \text{ kN}(x_2 - 5 \text{ m})$$

+ 5 kN/m(x₂ - 5 m)
$$\left(\frac{x_2 - 5 m}{2}\right)$$
 + $M = 0$

$$M = (-2.5x_2^2 + 15.75x_2 + 92.5) \text{ kN} \cdot \text{m}$$
 (4)

These results can be checked in part by noting that w = dV/dx and V = dM/dx. Also, when $x_1 = 0$, Eqs. 1 and 2 give V = 5.75 kN and $M = 80 \text{ kN} \cdot \text{m}$; when $x_2 = 10 \text{ m}$, Eqs. 3 and 4 give V = -34.25 kN and M = 0. These values check with the support reactions shown on the free-body diagram, Fig. 6–7d.

